

Politecnico di Bari

Dipartimento di Scienze dell'Ingegneria Civile e dell'Architettura



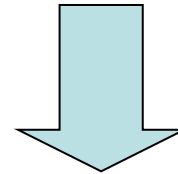
An Innovative solution for Water Bottling Using PET

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The idea



To pressurize PET bottles in order to balance an external axial load and to reduce the amount of utilized PET.

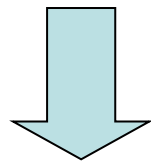


Two conditions:

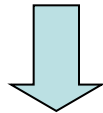
- 1) The bottle must not exceed the **elastic limit** of PET.
- 2) The geometry must not reach the **geometric instability (buckling)**.

Main Physical Phenomenon

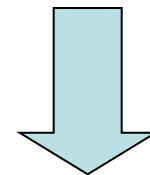
The deformed geometry depends on the applied forces, that is on the external load and the inner pressure of fluids (water and air).



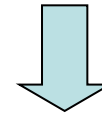
$$V=f(p_i)$$



Since the bottle is a closed system, the contained matter is constant. The pressure of fluids depends on the free volume and consequently on the deformed shape of the bottle.

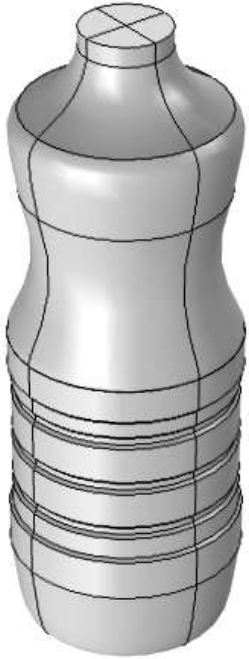


$$p_i=f(V)$$



Implicit System

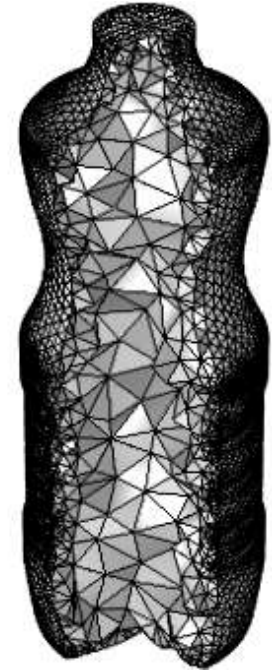
Geometry and meshes



Generic geometry
Height 190 mm
Diameter 67 mm



Mesh of surface
23 thousand triangles



Mesh of volume:
76 thousand tetrahedra

Mathematical models

- Solid phase:

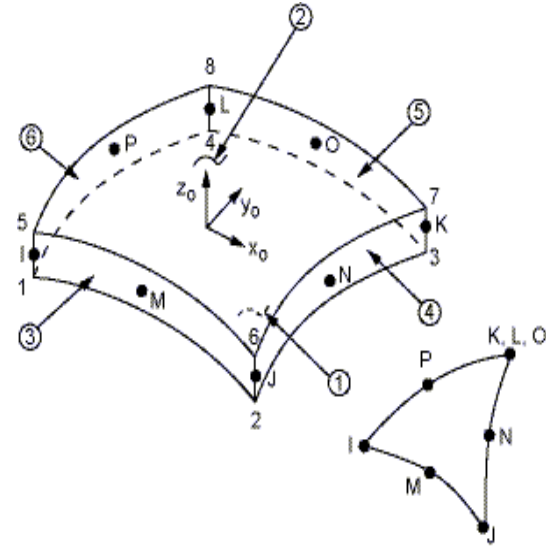
SHELL MODULE

With **geometric nonlinearities**

$d=0,167$ mm

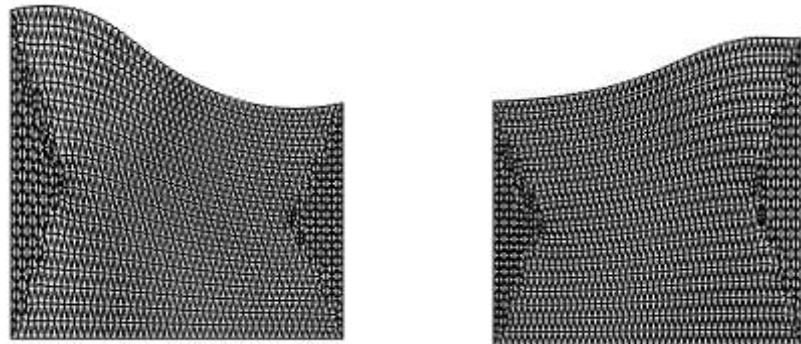
$D=67$ mm

$d/D=2,5 \cdot 10^{-3} \ll 1$

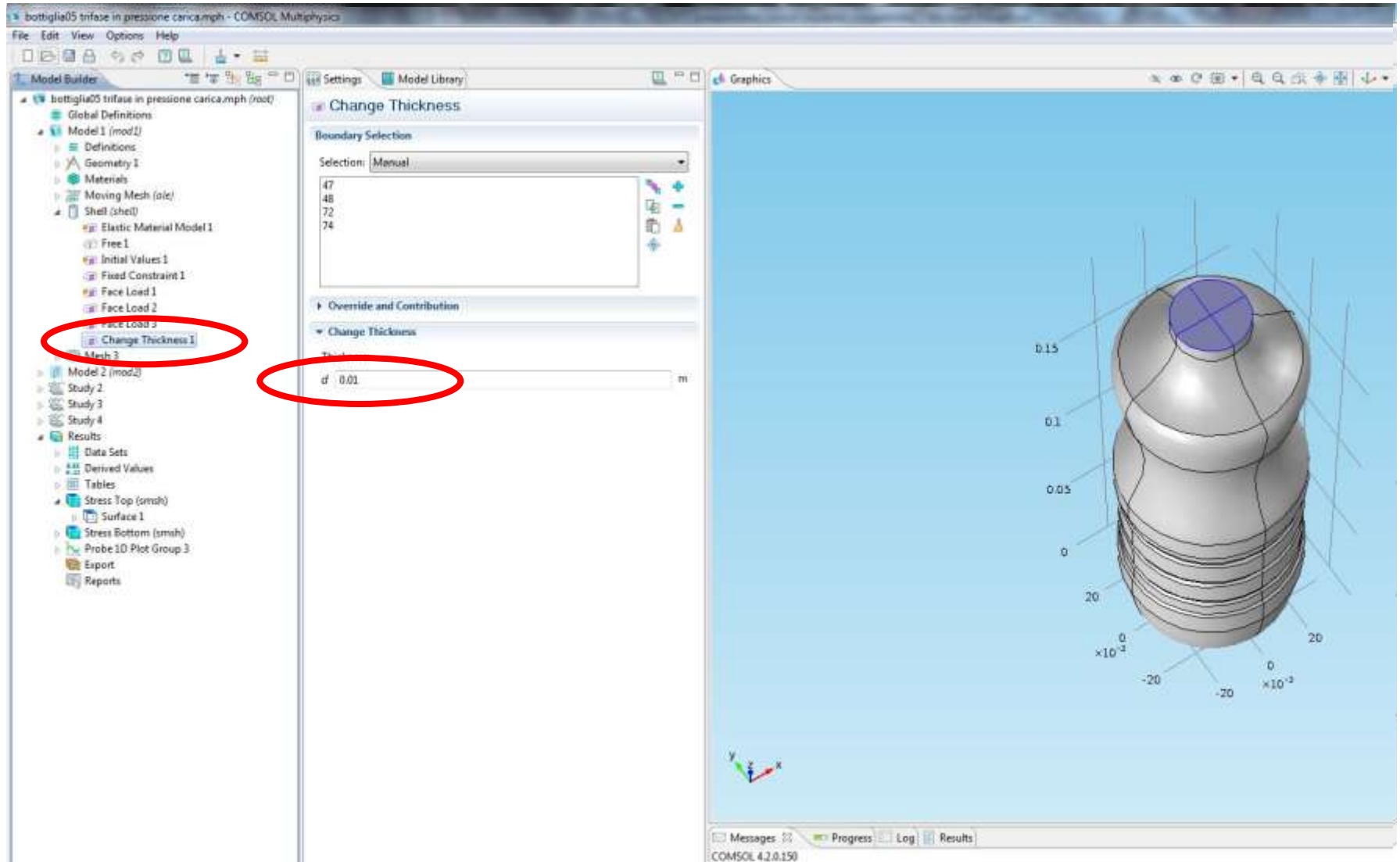


- Fluid phases:

MOVING MESH MODULE



SHELL Module

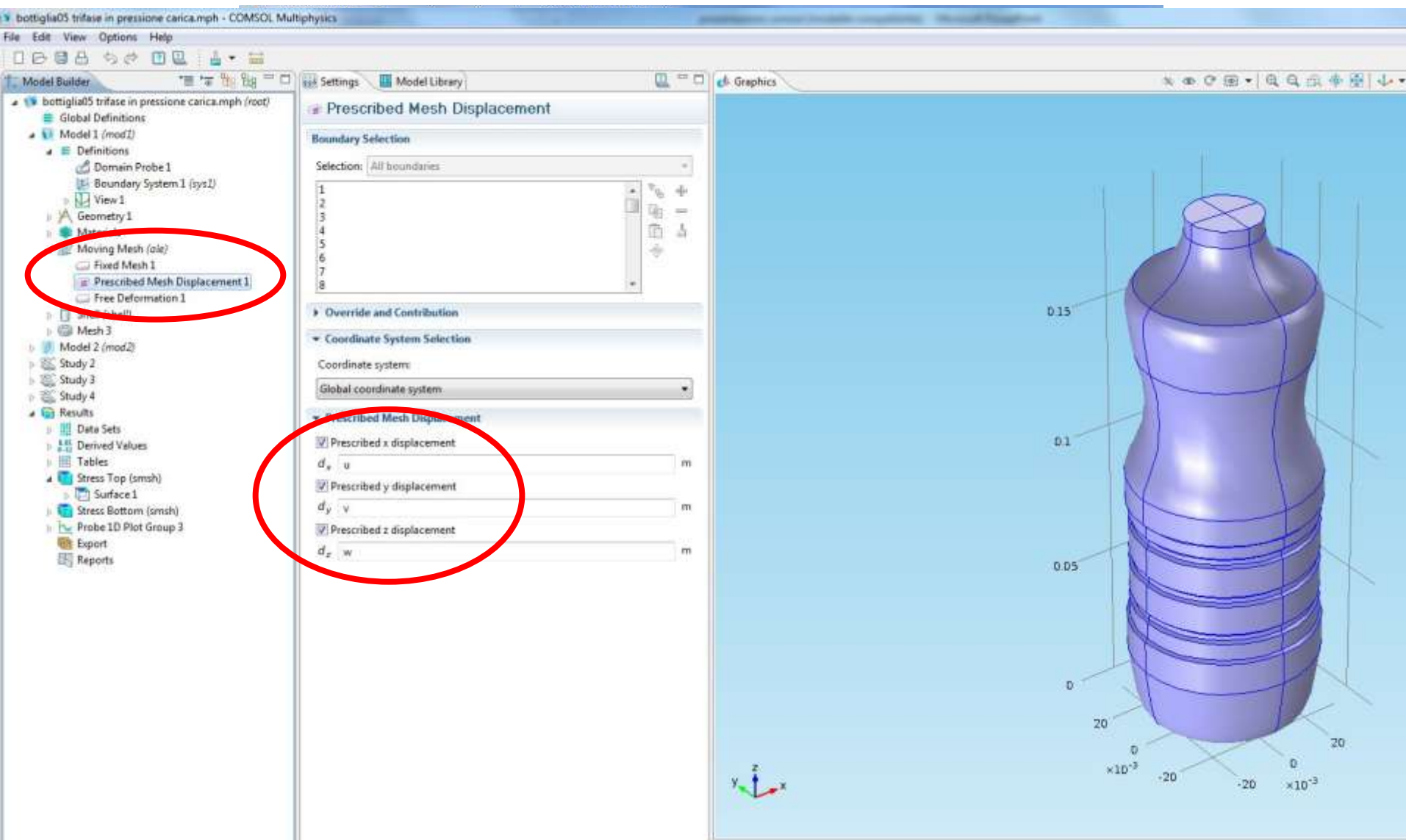


MOVING MESH Module

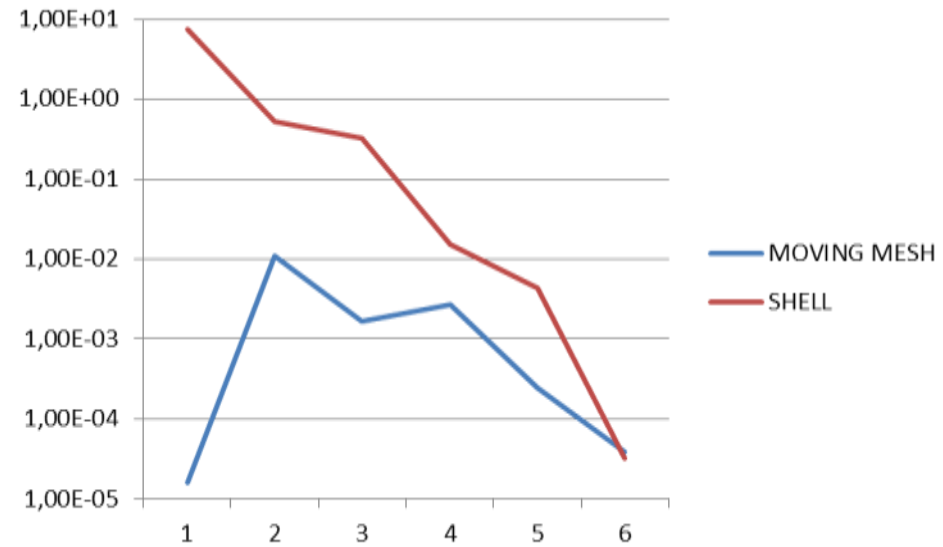
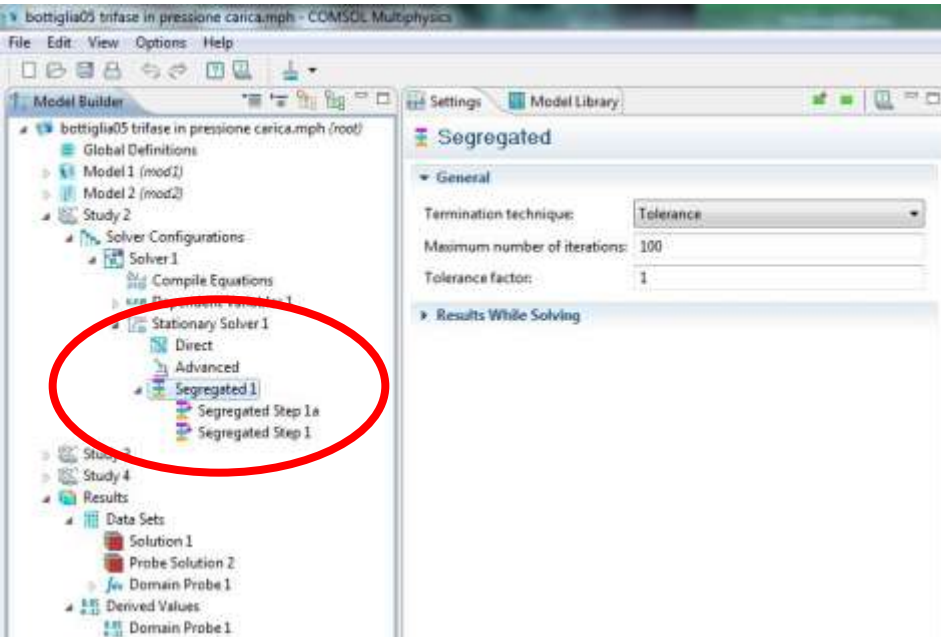
- Since water is incompressible, a change in its pressure does not affect its volume.
- As a consequence, the volume taken by the air is equal to the difference between the deformed shape of the bottle and the volume of the water.
- The volume of the bottle is evaluated by means of a “probe” defined as the integral of the unit over the volumetric mesh.
- The relative pressure of air is linked to the volume of the bottle by the following relationship:

$$p = p_i * \left(\frac{V_{i,gas}}{V_{bottle} - V_{water}} \right)^{1,4}$$

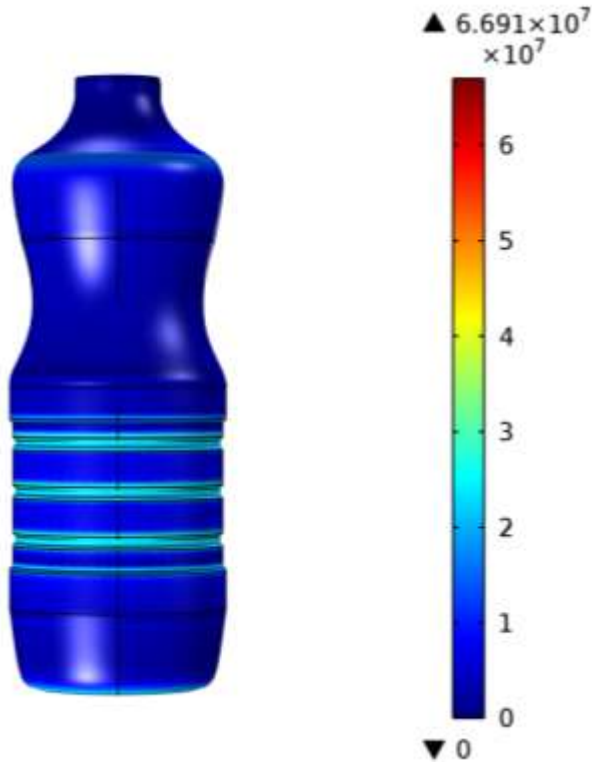
MOVING MESH Module



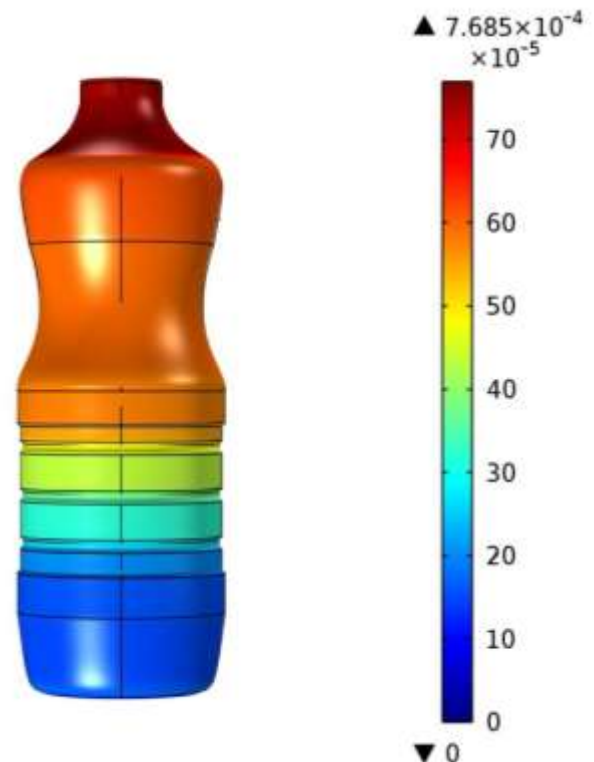
Solver



Results (without load)

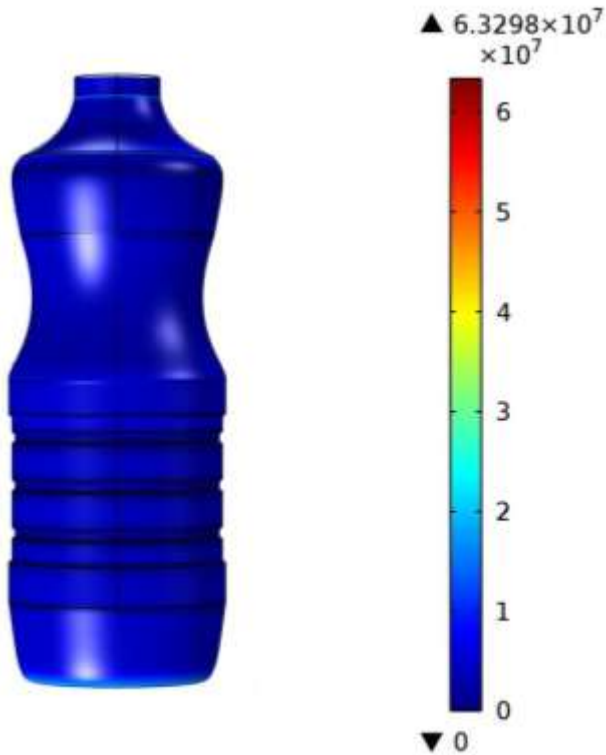


Von Mises stresses [Pa]

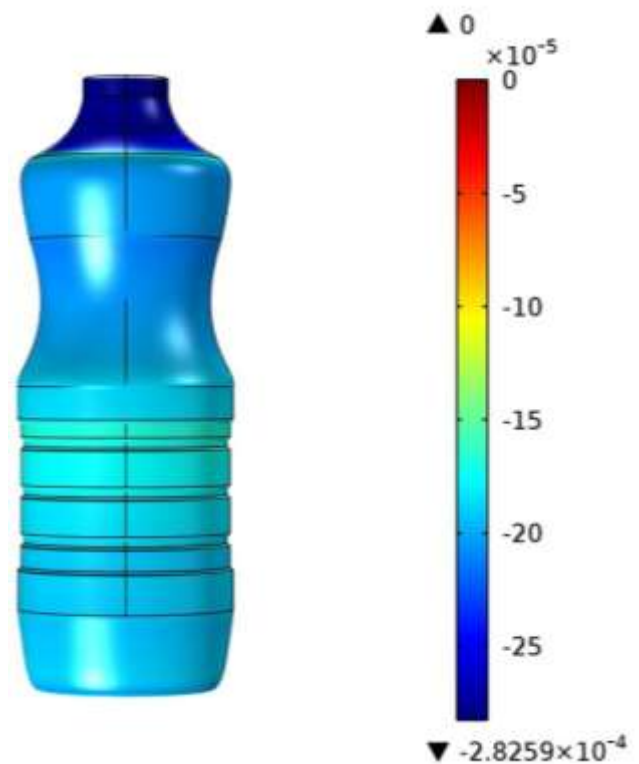


Displacement field,
axial component [m]

Results (with load)



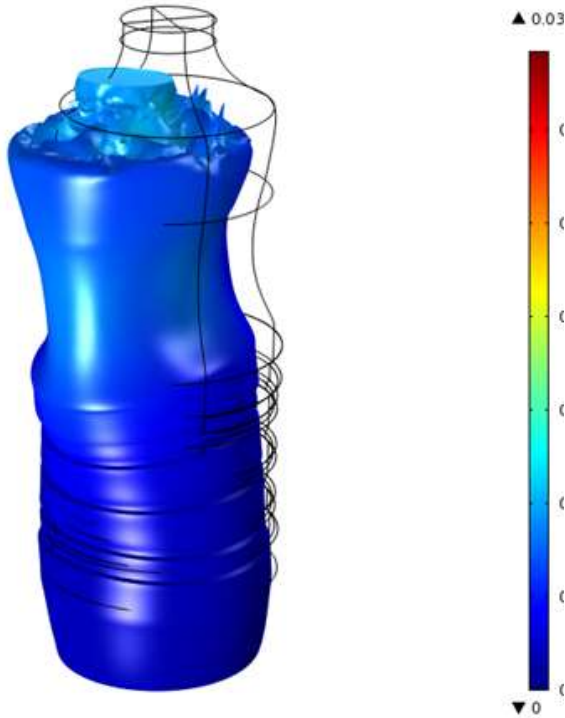
Von Mises stresses [Pa]



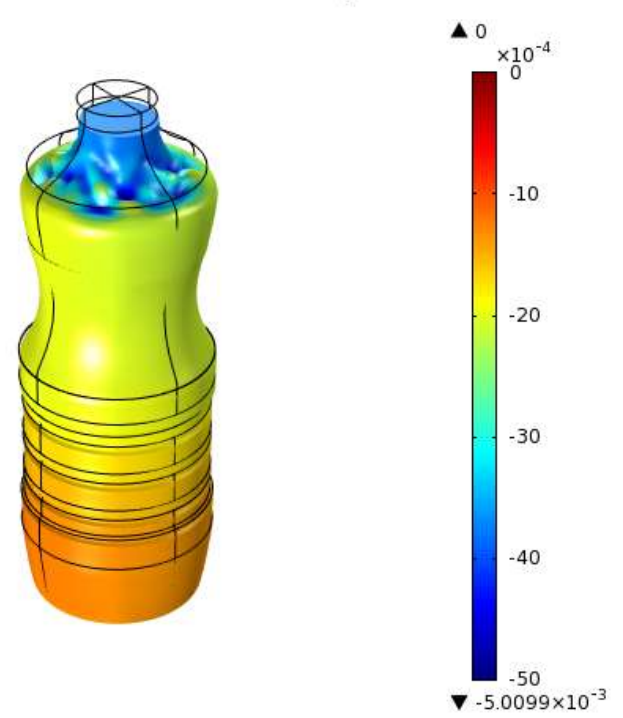
Displacement field,
axial component [m]

Results (buckling)

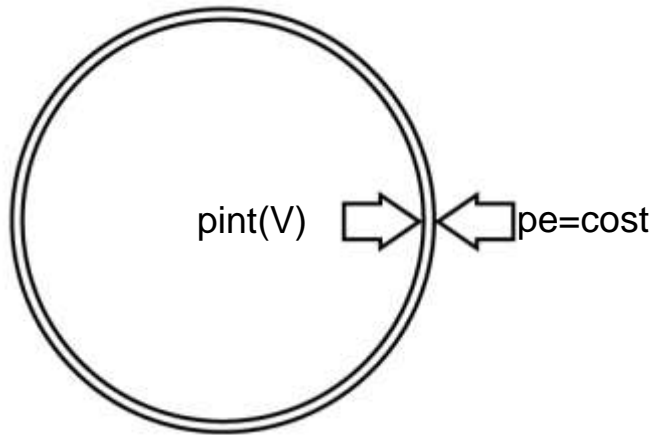
Surface: Total displacement (m) Surface Deformation: Displacement field (Material)



Surface: Displacement field, Z component (m) Surface Deformation: Displacement field (Material)



A test case: analytical solution



Diameter $r_o=1$ m

Thickness $s=1$ mm

$p_{int,0}= 6$ bar

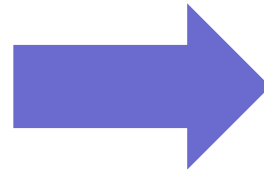
$p_e= 3$ bar

$E = 200$ GPa

$\nu= 0,29$

$$r_i = r_o + \frac{(p_{int,i} - p_e) * r_o^2}{2Es} (1 - \nu)$$

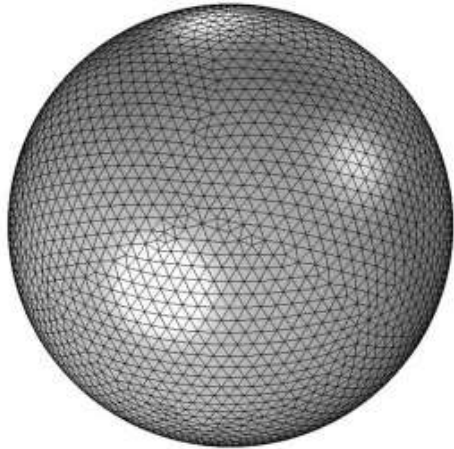
$$p_{int,i} = p_{int,1} * \left(\frac{V_1}{V_i}\right)^\gamma$$



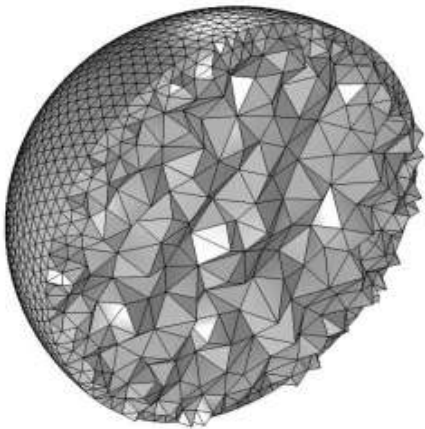
$r_i - r_o = 0,53487$ mm

$p_{int,i} = 601366$ Pa

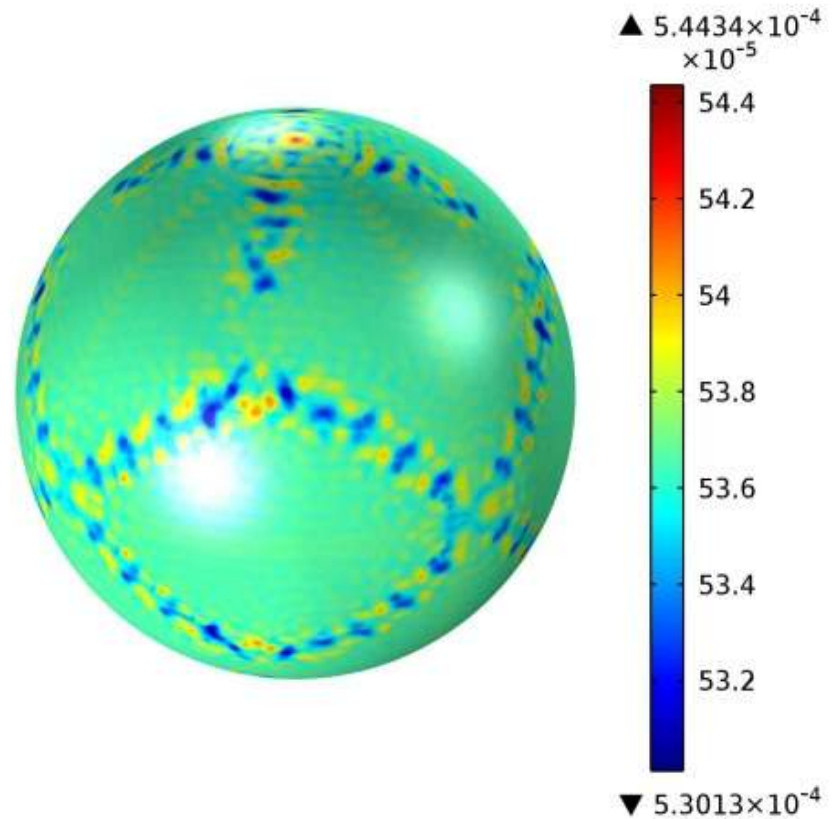
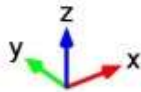
A test case: numerical solution



7000 triangles



37000 tetrahedra



Any questions?