



# Thickness Optimization of a Piezoelectric Converter for Energy Harvesting

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# Introduction

Extensive diffusion of sensing nodes and sensing networks (automotive, factory automation, entertainment, environment monitoring, security systems,...)

Main issue: **power supply**

**Batteries**

- ↓ • limited lifetime
- ↓ • recharging/replacement/disposal
- ↓ • cost



**Power harvesting**

- ↑ • in principle unlimited lifetime
- ↑ • unattended operation
- ↓ • sometimes limited output power

Environment

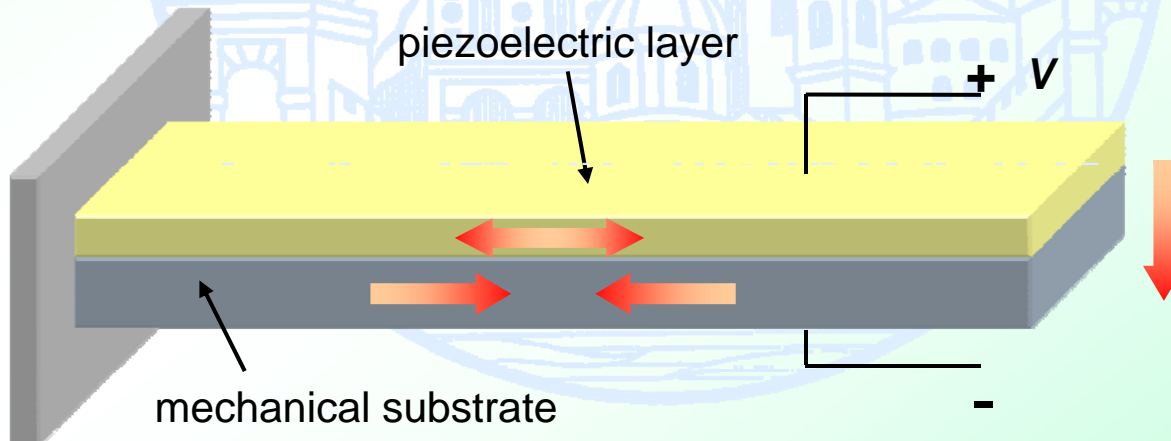
- ✓ Solar energy
- ✓ Mechanical vibrations
- ✓ Temperature gradients
- ✓ ...

# Mechanoelectrical energy conversion

- Conversion techniques
  - electromagnetic (inductive)
  - electrostatic (capacitive)
  - **piezoelectric**

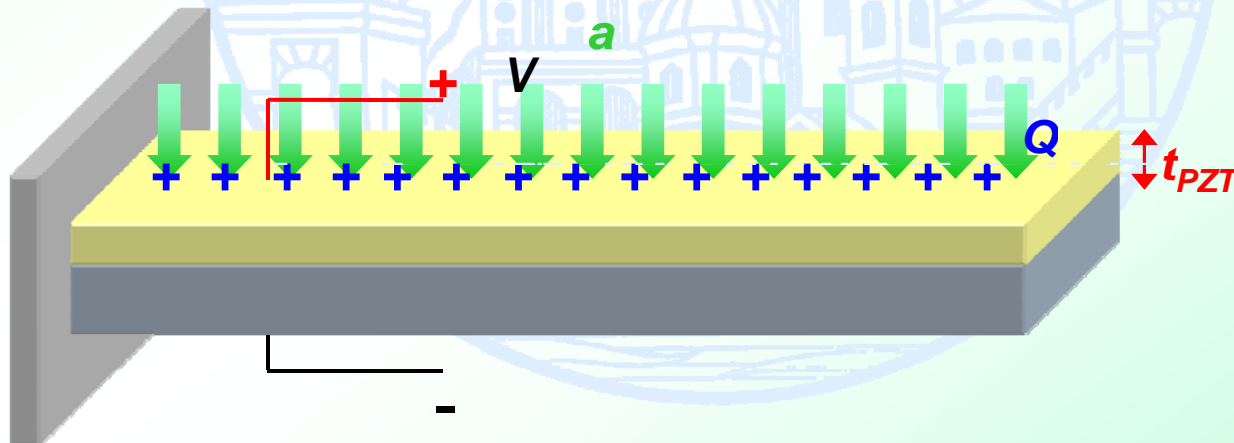
**Direct piezoelectric effect:** surface charge induced by a mechanical stress

- Piezoelectric energy converter
  - unimorph cantilever shape (thick-film, MEMS technologies)



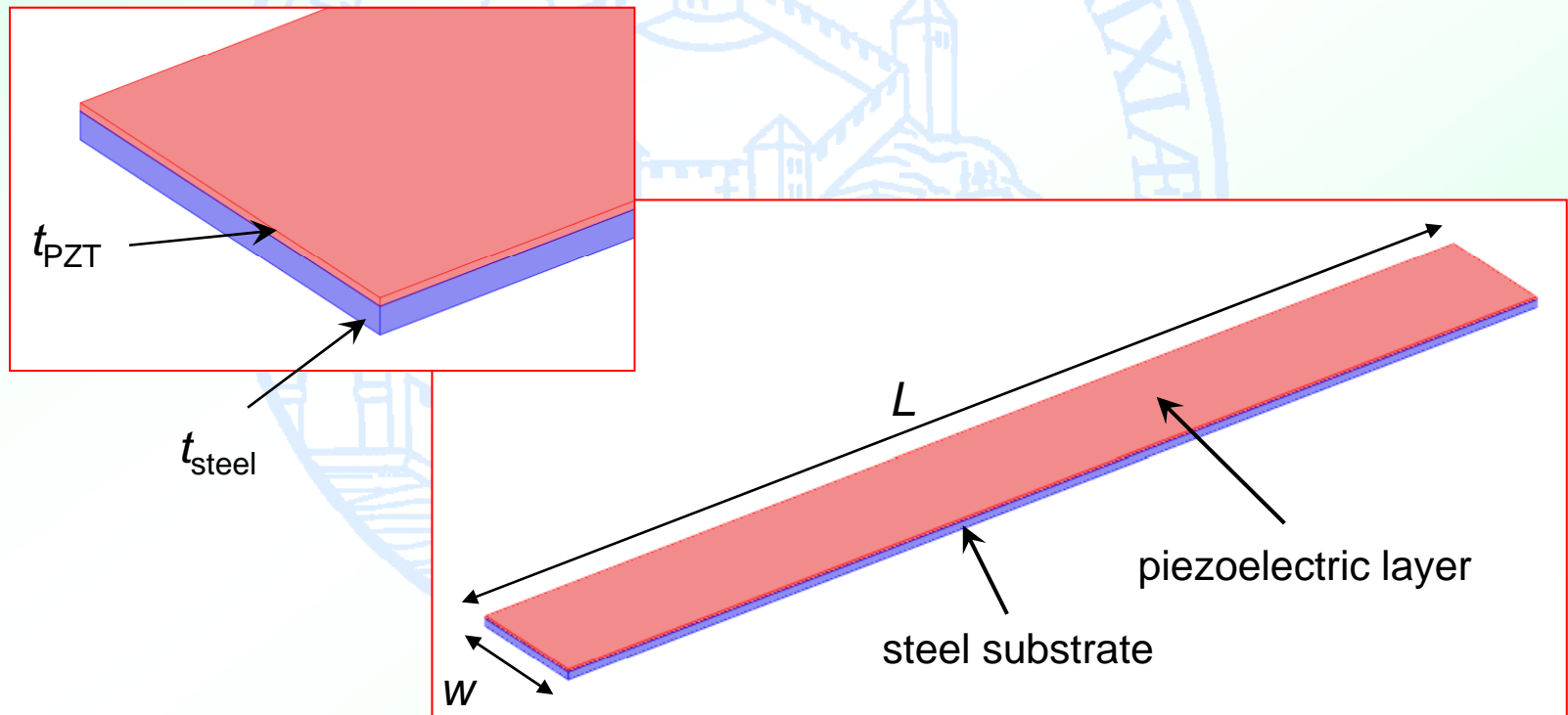
# Use of COMSOL

- Improvement of the converted energy
  - ↳ Optimization of the dimensions
    - piezoelectric layer thickness
- Application modes
  - piezoelectric: mechanical / electrical behaviour
    - sinusoidal vertical acceleration application
    - generated charge / voltage
  - moving mesh: thickness change



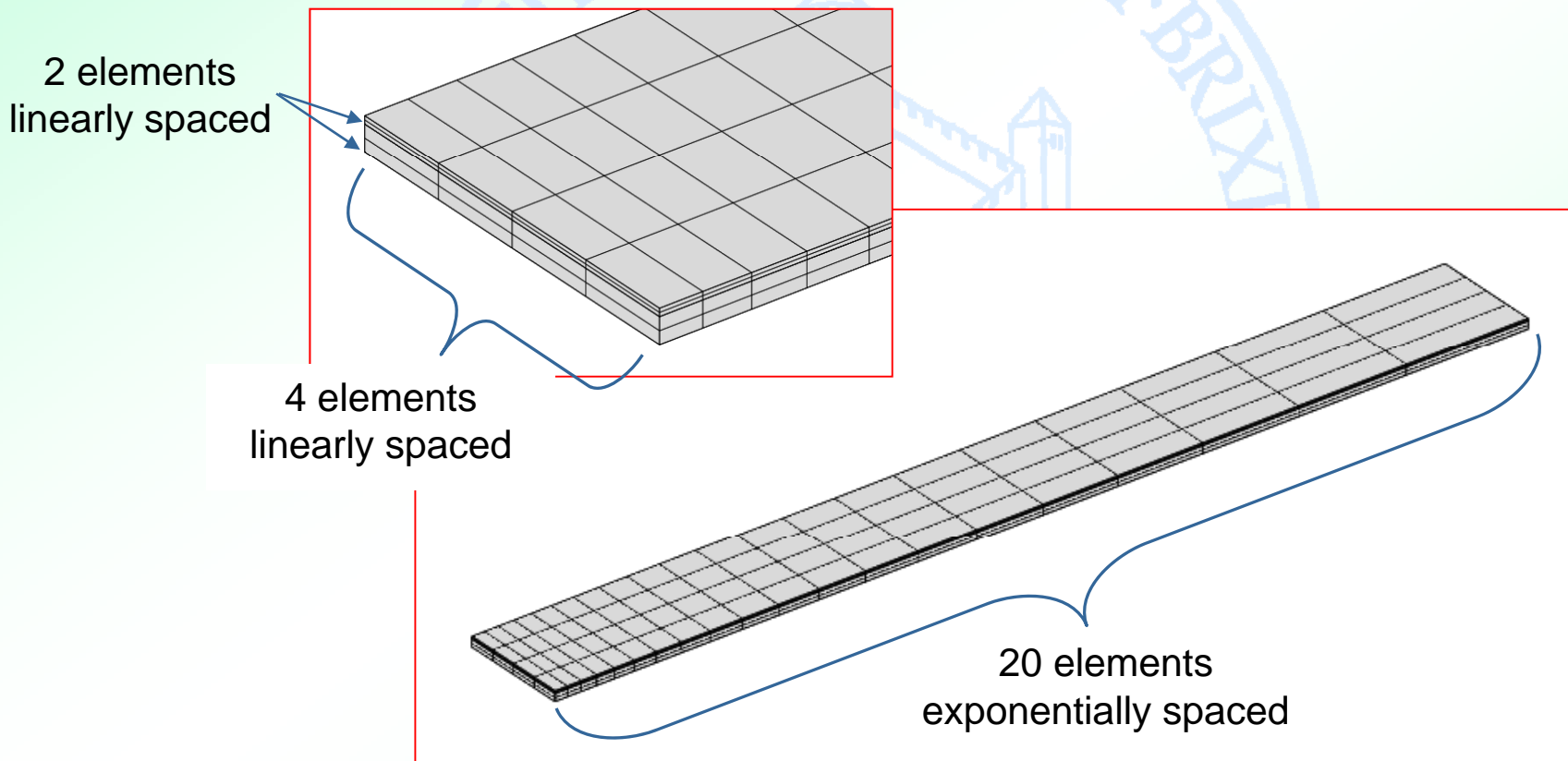
# Geometry

- 3D cantilever
  - length  $L = 27$  mm;
  - width  $w = 3$  mm;
  - steel thickness  $t_{\text{steel}} = 200$   $\mu\text{m}$ ;
  - piezoelectric layer thickness  $t_{\text{PZT}} = 60$   $\mu\text{m}$  (poled along thickness)



# Mesh

- Mapped mesh
  - 320 quad elements; 11808 degrees of freedom



# Governing equations

- Piezoelectric layer
  - strain-charge form

$$S = s^E T + dE$$

$$D = \varepsilon^T E + dT$$

- $S$  = mechanical strain
- $T$  = mechanical stress [N/m<sup>2</sup>]
- $s^E$  = elastic compliance [Pa<sup>-1</sup>]
- $d$  = piezoelectric coefficient [C/N]
- $D$  = electric displacement [C/m<sup>2</sup>]
- $E$  = electric field [V/m]
- $\varepsilon^T$  = dielectric permittivity [F/m]

- Steel layer

$$S = sT$$

$$s^E = \begin{bmatrix} 50 & -20 & -20 & 0 & 0 & 0 \\ -20 & 50 & -20 & 0 & 0 & 0 \\ -20 & -20 & 50 & 0 & 0 & 0 \\ 0 & 0 & 0 & 70 & 0 & 0 \\ 0 & 0 & 0 & 0 & 70 & 0 \\ 0 & 0 & 0 & 0 & 0 & 70 \end{bmatrix} \times 10^{-12} \text{ Pa}^{-1}$$

$$d = \begin{bmatrix} 0 & 0 & 0 & 0 & 11 & 0 \\ 0 & 0 & 0 & 11 & 0 & 0 \\ -2.5 & -2.5 & 5 & 0 & 0 & 0 \end{bmatrix} \times 10^{-12} \text{ CN}^{-1}$$

$$\varepsilon^T = \begin{bmatrix} 50 & 0 & 0 \\ 0 & 50 & 0 \\ 0 & 0 & 50 \end{bmatrix} \times \varepsilon_0$$

$$\rho = 3000 \text{ kg} \cdot \text{m}^{-3}$$

$$s = 5 \times 10^{-12} \text{ Pa}^{-1}$$



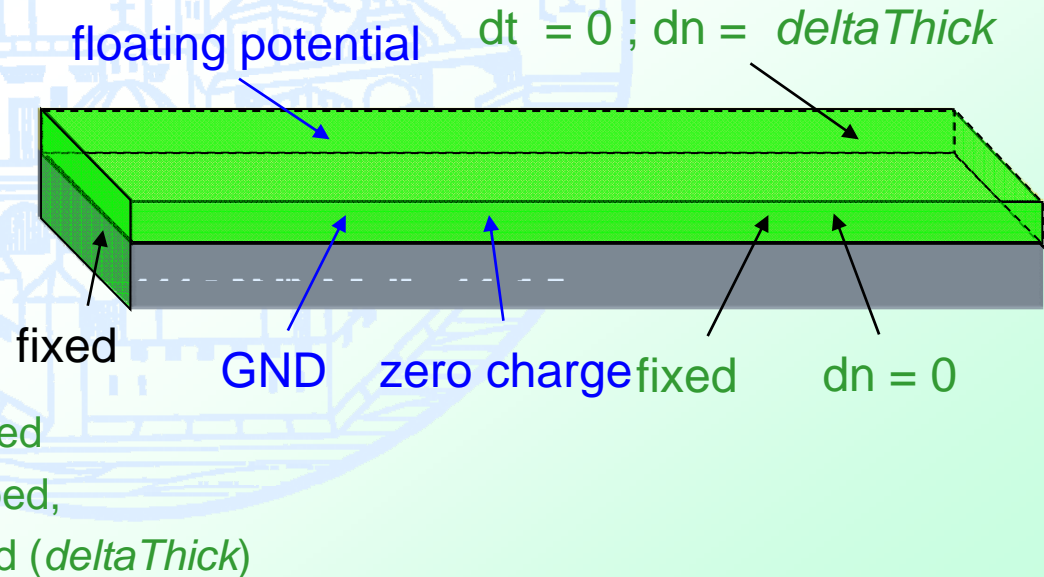
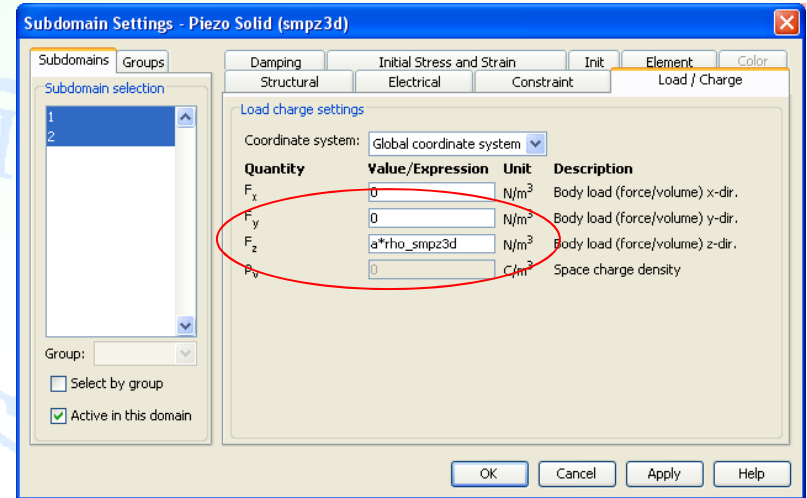
# Subdomain and boundary conditions

- Vertical acceleration
  - Body load  $F_z = a\rho$ 
    - $a = 0.1 g$
    - $\rho = \text{material density}$

- Mechanical boundary conditions
  - clamped end

- Electrostatic boundary conditions
  - bottom surface: *grounded*
  - upper surface: *floating potential*
  - other surfaces: *zero charge*

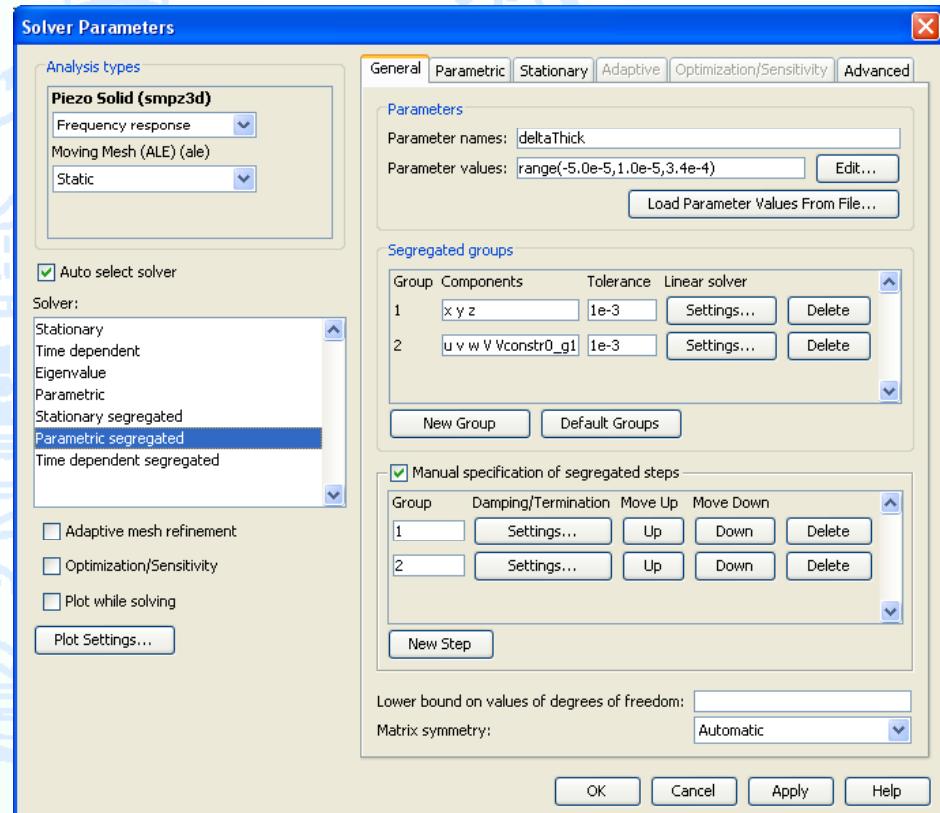
- Mesh boundary conditions
  - bottom surface: *clamped*
  - vertical surfaces: *normally clamped*
  - upper surface: *tangentially clamped, normally displaced ( $\delta Thick$ )*



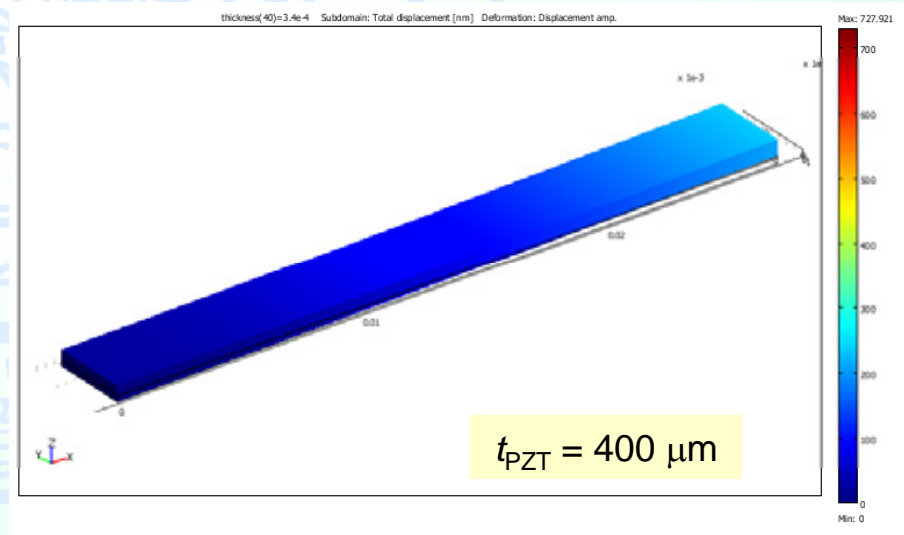
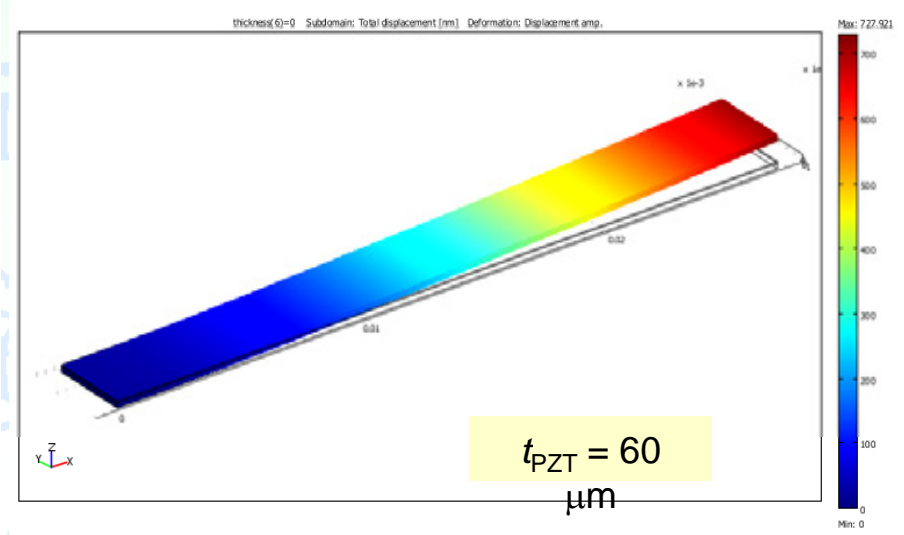
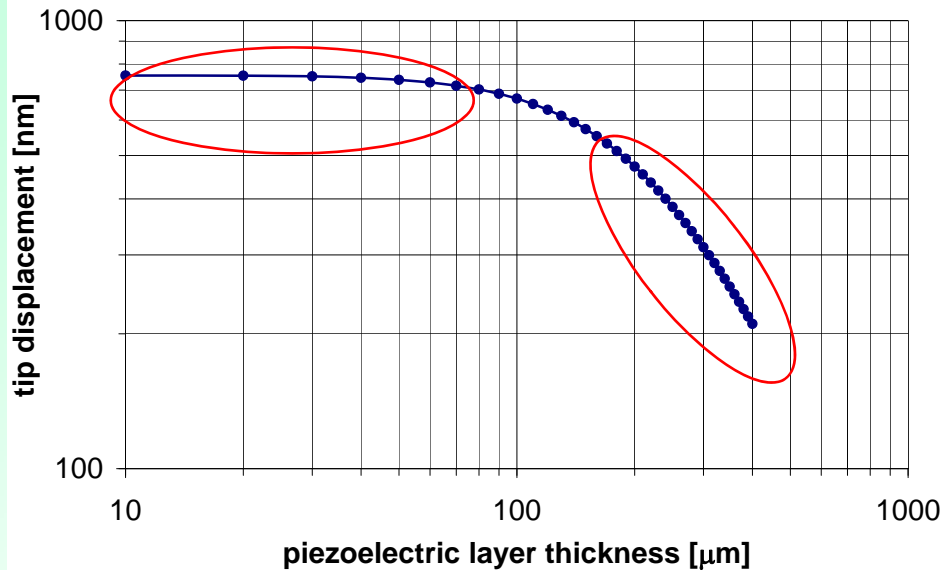


# Solver parameters

- Parametric segregated solver
  - group 1: moving mesh, static analysis
    - *deltaThick*:  $-50 \mu\text{m} \rightarrow 340 \mu\text{m}$
  - group 2: piezoelectric variables, frequency response
    - frequency = 10 Hz



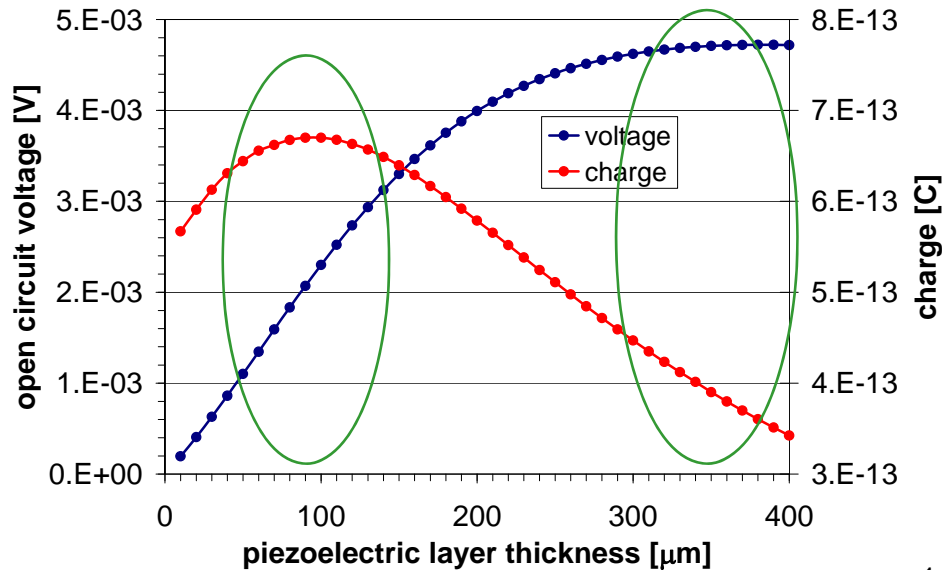
# Results – tip displacement



PZT rigidity  $\ll$  steel rigidity  
tip displacement  $\approx$  const

PZT rigidity  $\gg$  steel rigidity  
tip displacement  $\approx \frac{1}{t_{\text{PZT}}^n}$

# Results – electrical output

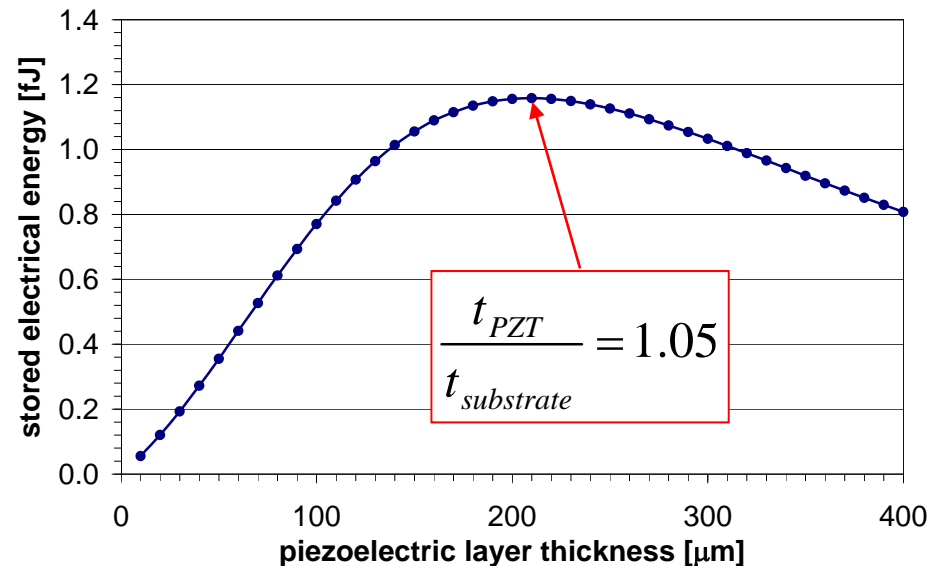


PZT rigidity << steel rigidity  
charge: maximum  
voltage: increases

PZT rigidity >> steel rigidity  
charge: decreases  
voltage ≈ const

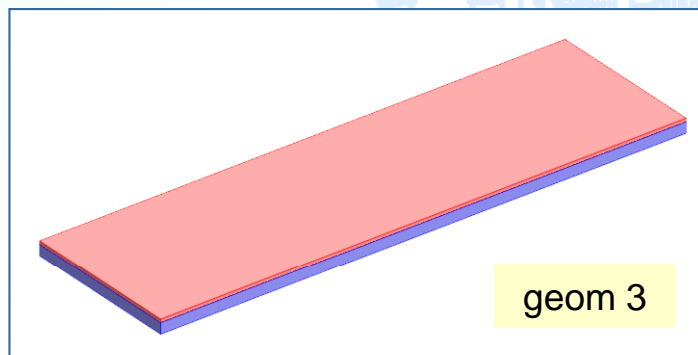
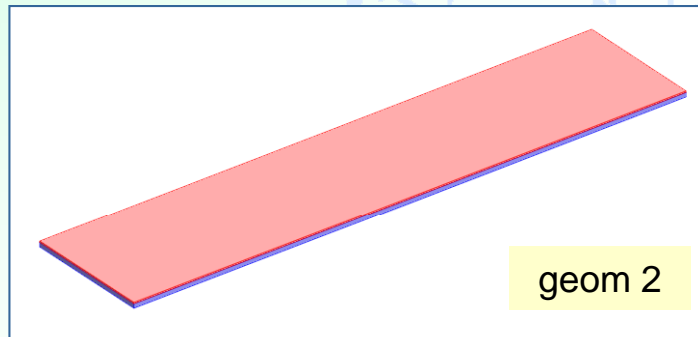
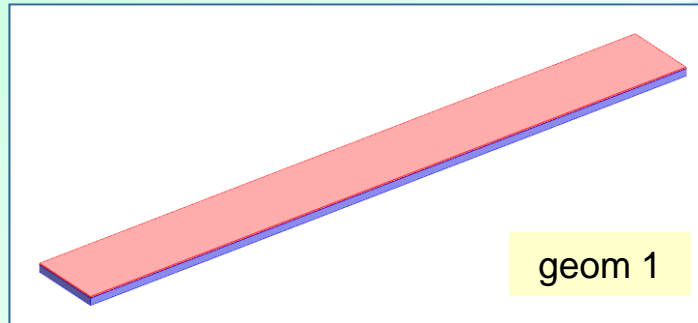
- Stored electrical energy

$$E = \frac{1}{2} QV$$

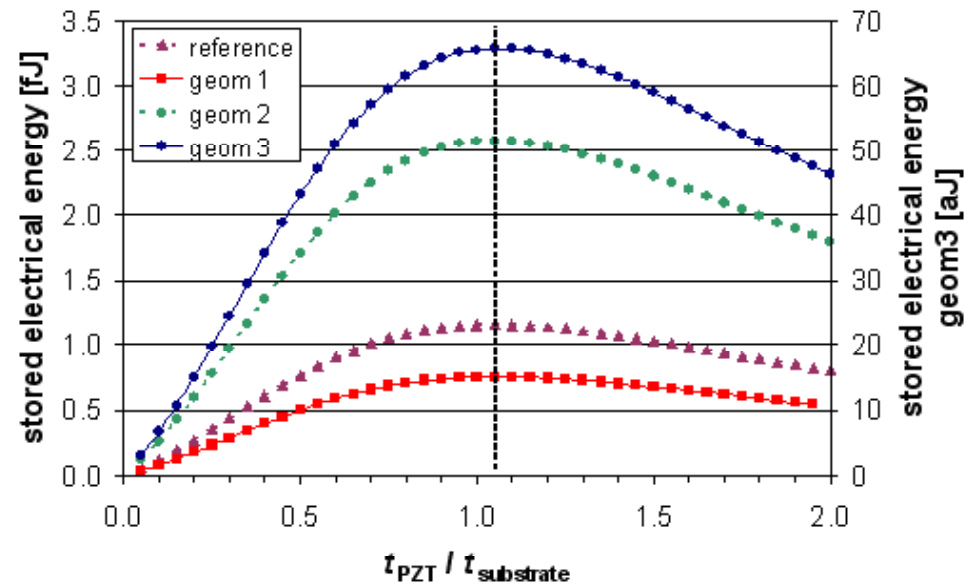


# Results – influence of the geometry

- Different geometries



	Substrate thickness [μm]	Width [mm]	Length [mm]
reference	200	3	27
geometry 1	300	3	27
geometry 2	200	6	27
geometry 3	200	3	10



# Conclusions

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- FEM simulations used for optimizing the geometrical dimensions of a piezoelectric energy converter
- Geometry with parametrized thickness
  - piezoelectric application mode
  - moving mesh application mode
- The optimal  $t_{\text{PZT}}/t_{\text{substrate}}$  was found for maximizing the electrical energy
- The optimal  $t_{\text{PZT}}/t_{\text{substrate}}$  value is independent from the converter dimensions
- This model is a specific example of using moving mesh for device geometry optimization